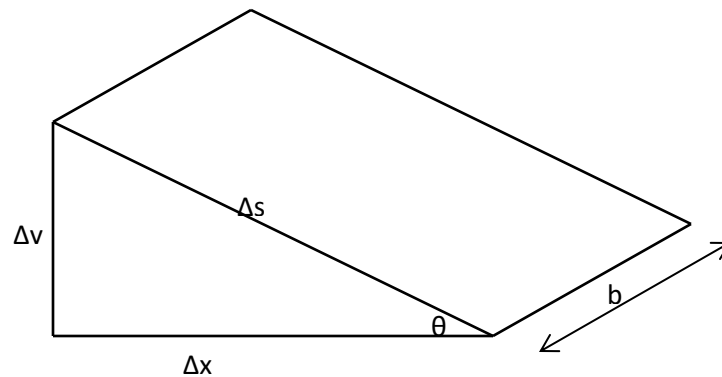


Pressure distribution in a fluid:

- ✓ There are many instances where the fluid is in stationary condition. That is the movement of liquid (or gas) is not involved.
- ✓ Yet, we have to solve some engineering problems involving stationary liquids (similar to what you have studied in statics of the course Engineering Mechanics).
- ✓ The condition of fluid, for such case is termed as hydrostatic.
- ✓ Recall, a fluid at rest cannot resist shear.
- ✓ Therefore, the normal stress on any plane through a fluid element at rest is called fluid pressure, p. (Recall you have studied normal stress & shear stress in Solid Mechanics course).
- ✓ This normal stress, when fluid is at rest, is taken positive for compression (conventionally taken through & across various course).
- ✓ To describe about this pressure, let us describe through equilibrium of forced in a small wedge of fluid at rest. (again recall principles of statics)



- ✓ For a body in static condition you should satisfy

$$\sum F_x = 0$$

$$\sum F_y = 0$$

$$\sum F_z = 0$$
- ✓ The above fluid element is also in static equilibrium.
- ✓ As this element is isolated from its surroundings, we need to incorporate corresponding forces (Free body diagram).

- ✓ By convention the compressive forces may be acting on the wall of the fluid element, i.e., forces act into the plane in respective directions.
- ✓ Let p_x be the force per unit area acting on the plane $b\Delta z$.
- ✓ Let p_z be the force per unit area acting on the plane $b\Delta x$.
- ✓ Let p_y be the force per unit area acting on the plane $\Delta x\Delta z$.
- ✓ Let p_s be the force per unit area acting on the plane $b\Delta s$.

For $\sum F_x = 0$;

$$p_x b\Delta z - p_s b\Delta s \sin \theta = 0 \dots\dots\dots (1)$$

For $\sum F_z = 0$;

$$p_z b\Delta x - p_s b\Delta s \cos \theta - \frac{1}{2} \rho g \Delta x \Delta z b = 0 \dots\dots\dots (2)$$

For $\sum F_y = 0$;

$$-\frac{1}{2} p_y \Delta x \Delta z + \frac{1}{2} p_y \Delta x \Delta z = 0 \dots\dots\dots (3)$$

From (1),

$$\begin{aligned} p_x b\Delta z - p_s b\Delta s \sin \theta &= 0 \\ p_x b\Delta z &= p_s b\Delta z \\ p_x &= p_s \end{aligned}$$

From (2),

$$\begin{aligned} p_z b\Delta x - p_s b\Delta x - \frac{1}{2} \rho g \Delta x \Delta z b &= 0 \\ p_z &= p_s + \frac{1}{2} \rho g \Delta z \end{aligned}$$

For limit ($\Delta z \rightarrow 0$; $\Delta x \rightarrow 0$) $p_z \rightarrow p_s$

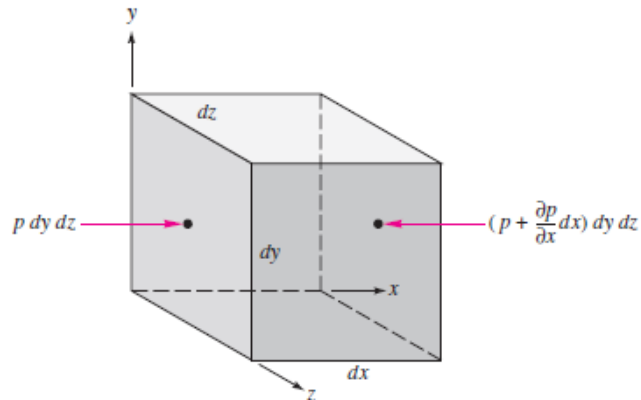
So, $p_z = p_s = p_x = p_y = p = \text{pressure}$

At a static point, pressure is a scalar property without any orientation. $P(x,y,z,t)$

Pressure force on fluid element:

Due to pressure, forces act on the respective plane of interest.

Let us consider a rectangular elemental prism of volume $\Delta x \Delta y \Delta z$



Net x force on an element due to pressure variation

(Source: Fluid Mechanics by F.M. White)

- ✓ Force due to pressure is called pressure force.
- ✓ At any mathematical point pressure is described as $p(x,y,z,t)$ as it is a scalar quantity.
- ✓ Therefore, let us suggest that in x direction there are two planes normal.
- ✓ On the left plane, let p be the pressure.
- ✓ On the right plane, pressure will be $p + \frac{\partial p}{\partial x} \Delta x$

So, net force due to pressure acting in the x direction will be:

$$\begin{aligned}\Delta F_{px} &= p \Delta y \Delta z - \left(p + \frac{\partial p}{\partial x} \Delta x \right) \Delta y \Delta z \\ &= - \frac{\partial p}{\partial x} \Delta x \Delta y \Delta z\end{aligned}$$

Similarly, net force in y direction:

$$\Delta F_{py} = - \frac{\partial p}{\partial y} \Delta x \Delta y \Delta z$$

Similarly, net force in z direction due to pressure

$$\Delta F_{pz} = -\frac{\partial p}{\partial z} \Delta x \Delta y \Delta z$$

Force is a vector & the net pressure force is given by,

$$\begin{aligned}\overline{\Delta F}_p &= \Delta F_{px} \hat{i} + \Delta F_{py} \hat{j} + \Delta F_{pz} \hat{k} \\ &= \left[-\frac{\partial p}{\partial x} \hat{i} - \frac{\partial p}{\partial y} \hat{j} - \frac{\partial p}{\partial z} \hat{k} \right] \Delta x \Delta y \Delta z\end{aligned}$$

As the volume $\Delta x \Delta y \Delta z$ is arbitrary & chosen by us,

the net pressure force per unit volume

$$\begin{aligned}\vec{f}_p &= \left[-\frac{\partial p}{\partial x} \hat{i} - \frac{\partial p}{\partial y} \hat{j} - \frac{\partial p}{\partial z} \hat{k} \right] \\ &= \nabla p \\ &= \text{gradient of pressure}\end{aligned}$$

i.e., it is the gradient of pressure that cause force in the fluid.

Gage Pressure & Vacuum Pressure:

In the physics classes, you might have already seen how pressure is expressed.

- ✓ The units are N/m^2 or Pa or kPa etc.
- ✓ Might have heard about
 1. Absolute pressure,
 2. Gage pressure,
 3. Vacuum pressure

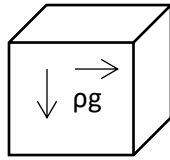
Hydrostatic Pressure Condition:

- If a fluid is in rest, it will not have any shear force.
- Similarly, in a static liquid, there will not be any viscous force.
- For the same rectangular prism element of the fluid in rest (or static):

We should apply principles of statics

$$\sum F = 0$$

- As the fluid is in static, the forces acting on it will be
 - pressure force
 - gravity force



$$\vec{g} = 0\hat{i} + 0\hat{j} + g\hat{k}$$

i.e. $\sum F_x = 0$ means $-\frac{\partial p}{\partial x} \Delta x \Delta y \Delta z = 0$

Or, $\frac{\partial p}{\partial x} = 0$

Similarly, $\sum F_y = 0$ implies $\frac{\partial p}{\partial y} = 0$

Similarly, $\sum F_z = 0$ means $-\frac{\partial p}{\partial z} \Delta x \Delta y \Delta z - \rho g_z \Delta x \Delta y \Delta z = 0$

Or, $\frac{\partial p}{\partial z} = -\rho g_z$

It can be also expressed as,

$$\vec{\nabla} p = 0, \quad -\nabla p \Delta x \Delta y \Delta z + \rho \vec{g} \Delta x \Delta y \Delta z = 0$$

$$\vec{\nabla} p = \rho \vec{g}$$

i.e., $\frac{\partial p}{\partial z} = -\rho g$

or $\vec{\nabla} p = \rho \vec{g}$

In hydrostatic condition, it is now clear that,

$$\frac{\partial p}{\partial x} = 0 \quad \text{and} \quad \frac{\partial p}{\partial y} = 0$$

That is, there won't be variation of pressure in horizontal direction for the same fluid.

So, we can hence write $\frac{\partial p}{\partial z}$ as $\frac{dp}{dz}$ in static conditions.

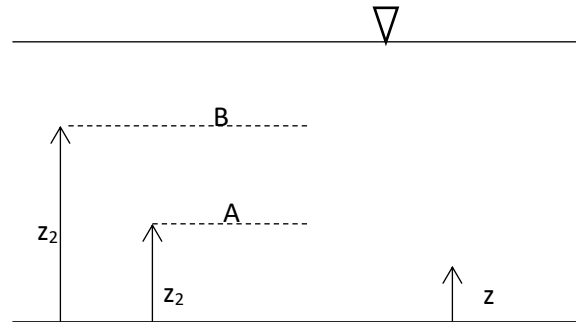
$$\frac{dp}{dz} = -\rho g_z$$

$$dp = -\rho g_z dz$$

$$\int_A^B dp = -\int_A^B \rho g_z dz$$

$$p_B - p_A = -\rho g_z (z_2 - z_1)$$

$$p_A = p_B + \rho g_z (z_2 - z_1)$$



If B is water surface, then $P_B = \text{atmospheric pressure} = 0$ (gage pressure)

So, $p_A = \rho g_z (z_2 - z_1) = \rho gh$, where $h = \text{height of water surface level from surface}$.